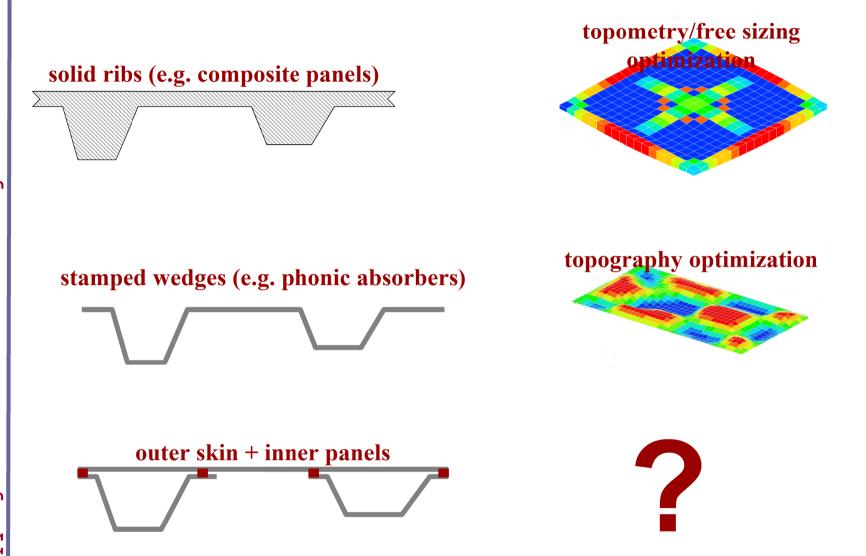


OPTIMAL INNER PANEL DESIGN

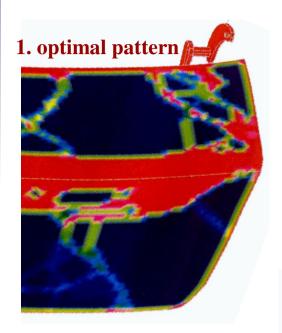


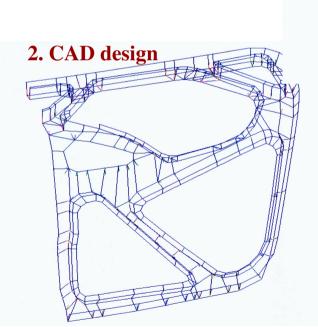
Problem: what is the appropriate optimization model for inner panels?

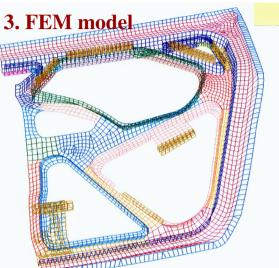




Inner panel design is usually based on solid rib patterns obtained from topology or topometry optimization







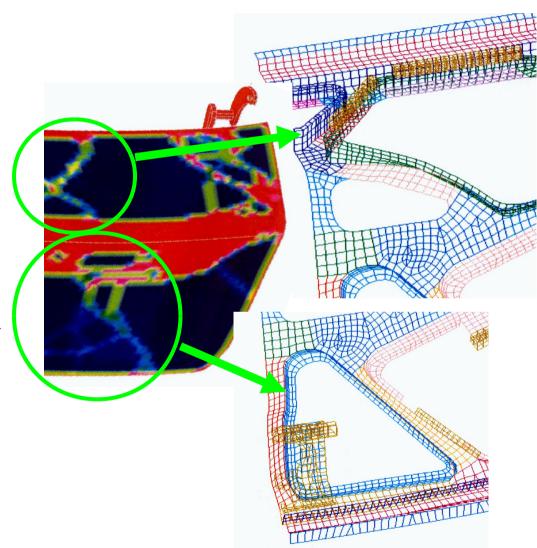
Problem: a solid rib is different than a rib of a inner panel:

•It is stiffer

•It is isotropic

As a result:

- •Shallow solid ribs become thick inner panel ribs
- •Sometimes, long shallow ribs must be redesigned with traditional methods

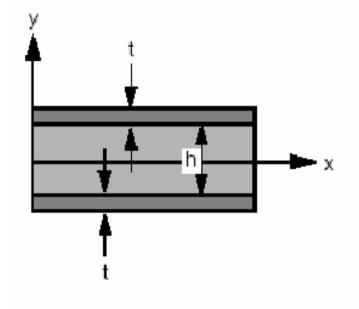




SimTech has developed an inner panel model suitable for topometry optimization.

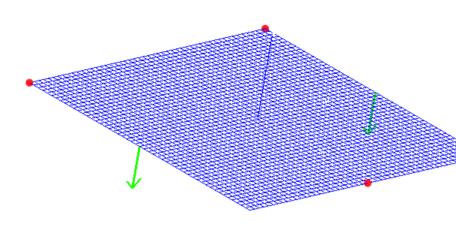
The design variable is h, the local height of the rib.

The behavior changes when h goes to zero, and the bottom panel disappears.





Example: plate under lateral flexural loads (simplified car hood)



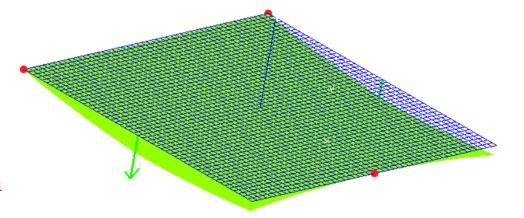
3 SPCs (2 hinges, 1 lock)

2 100 N loads

Material: 1.25 mm Aluminum

Target mid-span displacement : 2 mm

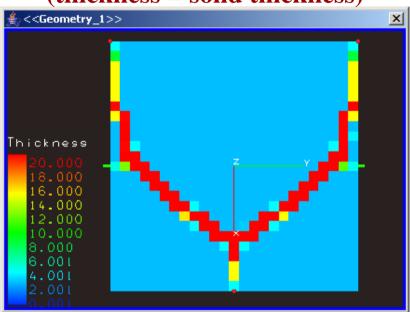
Displacement without reinforcements: 715 mm





Optimization results

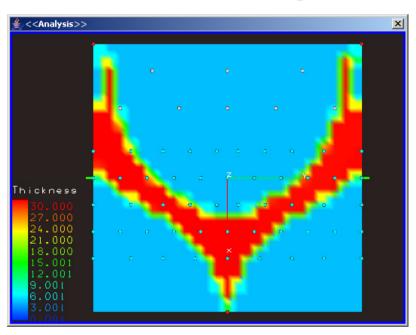
standard topometry (thickness = solid thickness)



Mid-span displacement = 2mm

Structure mass = 5.8 Kgs

inner panel formulation
(thickness = rib height)



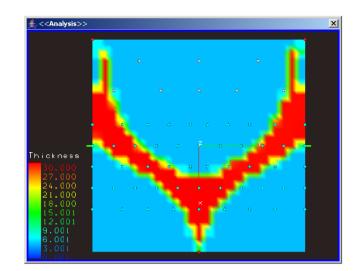
Mid-span displacement = 2 mm

Structure mass = 4.3 Kgs

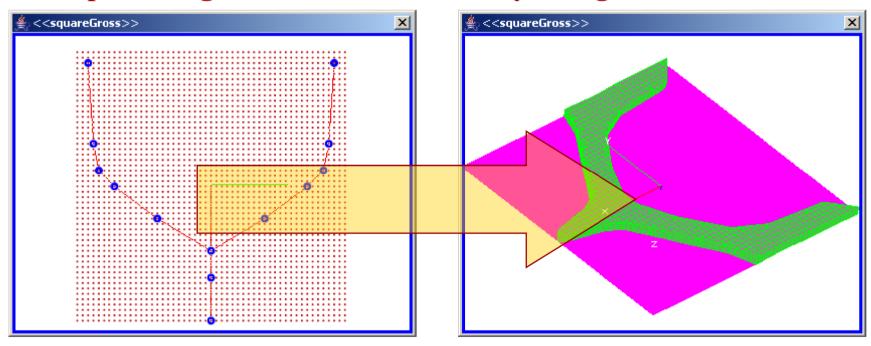


The result of the inner panel optimization is a good layout for a real inner panel.

To prove it, we generate an inner panel along this layout.



The panel is generated automatically using ENKIDOU tools

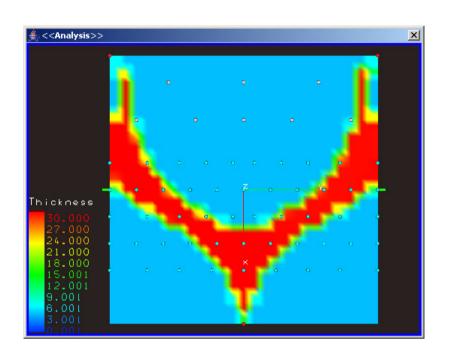


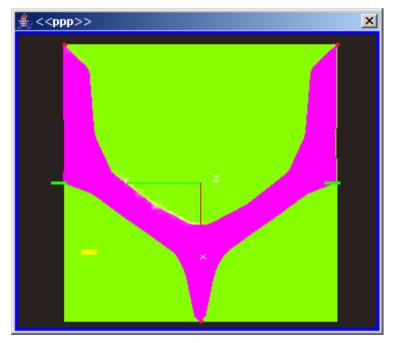
bar layout (manual generation)

inner panel (automatic generation)



The physical inner panel has the same size (width, height) of the numerical panel generated by GENESIS topometry





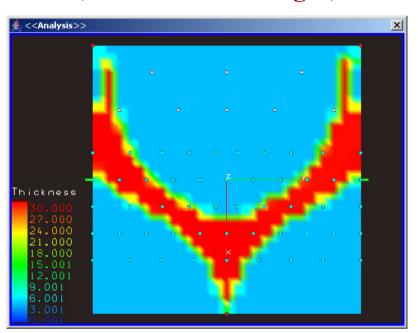
Rib height = 30 mm

Rib width: same for the two models



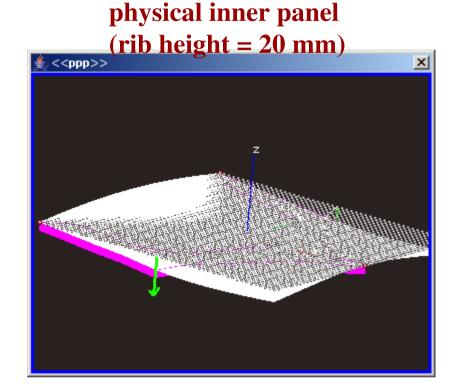
The physical panel (outer skin + inner panel) has the same behavior and mass of the single layer panel used in optimization.

inner panel formulation
(thickness = rib height)



Mid-span displacement = 2 mm

Structure mass = 4.3 Kgs



Mid-span displacement = 2.05 mm Structure mass = 4.7 Kgs

we can optimize directly inner panels for car bodies and other applications!